



ATTACHMENT B

SUBSTITUTE SPECIFICATION

(Showing All Changes Made to the Translated and Previously Amended Specification in Serial No. 10/510,140)

HYDRAULIC SYSTEM AND AUTOMATIC GEARBOX TRANSMISSION

CROSS-REFERENCE TO RELATED APPLICATION

This application is a continuation of International Application Serial No. PCT/DE03/01194, with an international filing date of April 10, 2003, and designating the United States, the entire contents of which is hereby incorporated by reference to the same extent as if fully rewritten.

BACKGROUND OF THE INVENTION

FIELD OF THE INVENTION

The present invention relates to a hydraulic system with a multi-flow hydraulic pressure supply unit, especially a dual-flow hydraulic pressure supply unit, such as a pump, through which a ~~volume~~ volumetric flow of hydraulic fluid is fed to a ~~consumer~~ hydraulic-fluid-operated device. The invention also ~~concerns~~ relates to an automatic transmission for motor vehicles.

DESCRIPTION OF THE RELATED ART

In modern motor vehicles, hydraulic systems in which at least one pressure supply unit supplies at least one ~~consumer~~ hydraulic-fluid-operated device with a defined pressure are being used to improve safety and comfort. The known hydraulic systems indeed have a high power density, a low power-weight ratio, and high dynamics, but they nonetheless require more energy in relation to regulated electric

drives, which leads to higher fuel consumption. Previously, mostly single-flow pumps have been used to supply pressure to, for example, automatic transmissions. Single-flow here means that the pump conveys one pump flow. In contrast, multi-flow pumps convey several pump flows independently of one another. The pump flows are thus connected in parallel.

~~The~~ An object of the present invention is to reduce the losses in known hydraulic systems. ~~Here the~~ The hydraulic pressure supply unit used in the hydraulic system should meet the demands of the automotive industry over a wide range of hydraulic requirements.

SUMMARY OF THE INVENTION

The object is ~~accomplished in~~ achieved by a hydraulic system with a multi-flow hydraulic pressure supply unit, especially a dual-flow, hydraulic pressure supply unit, such as a pump, through which a ~~volume~~ volumetric flow is fed to a ~~consumer, in that~~ a hydraulic-fluid-operated device. A valve ~~apparatus~~ for switching between the individual pump flows and/or for interconnecting the individual pump flows is provided. The valve ~~apparatus~~ enables actuating the individual pump flows selectively. It is possible ~~as-needed~~ to actuate only one or several pump flows with a single valve ~~apparatus~~ , as needed.

A preferred embodiment of the hydraulic system is characterized in that the individual pump flows are joined or separated through a ~~step~~ check valve. The ~~step~~ check valve makes it possible to ~~carry~~ conduct away at least one of the pump flows so

that optionally only at least one of the pump flows is conveyed to the ~~consumer~~ hydraulic-fluid-operated device.

A further preferred embodiment of the hydraulic system is characterized in that the at least one pump flow, which is separated by the ~~stop~~ check valve from the at least one other pump flow, can be ~~carried~~ conducted away through the valve apparatus. Only one pump flow or several pump flows are supplied to the ~~consumer~~ hydraulic-fluid-operated device as a function of the setting of the valve apparatus. The ~~stop~~ check valve prevents all the pump flow flows from being ~~evacuated~~ conducted away.

A further preferred embodiment of the hydraulic system is characterized in that the valve apparatus ~~comprises a~~ includes an end surface ~~pre-stressed~~ that is biased by a spring ~~loaded apparatus, said~~ , which surface ~~being~~ is acted upon with the dynamic feedback pressure of ~~a feedback~~ leading from the ~~consumer~~ hydraulic-fluid-operated device to the input side of the hydraulic pressure supply unit. The use of the ~~feedback~~ dynamic feedback pressure for actuating the valve apparatus ~~guarantees~~ assures that the valve apparatus switches from a single-flow to an at least dual-flow conveyance of the hydraulic pressure supply unit when the ~~feedback~~ dynamic feedback pressure drops below a specified minimum value. If the ~~feedback~~ dynamic feedback pressure exceeds a specified ~~maximal~~ maximum value, the valve apparatus switches from an at least dual-flow to an at least single-flow ~~conveyance of~~ delivery from the hydraulic pressure supply unit.

A further preferred embodiment of the hydraulic system is characterized in that a hydraulic ~~resistor~~ resistance is arranged between the valve apparatus and the input

side of the hydraulic pressure supply unit. The hydraulic ~~resistor~~ resistance serves to generate the feedback dynamic feedback pressure for actuating the valve apparatus.

A further preferred embodiment of the hydraulic system is characterized in that the valve apparatus includes a 2/2 way valve that releases a connection provided between the output side of a pump flow and the input side of the hydraulic pressure supply unit in the one position, ~~said~~ the connection being interrupted in the other position of the 2/2 way valve. The 2/2 way valve makes it possible to feed the two pump flows to the ~~consumer~~ hydraulic-fluid-operated device individually or together, depending upon need.

A further preferred embodiment of the hydraulic system is characterized in that the valve apparatus has three shifting stages, whereby in the first shifting stage a cooling circuit is not supplied and only one pump flow is conveyed to the ~~consumer~~ hydraulic-fluid-operated device by the hydraulic pressure supply unit, ~~whereby in~~ In the second shifting stage the cooling circuit is not supplied and at least two pump flows are conveyed to the ~~consumer~~ hydraulic-fluid-operated device from the hydraulic pressure supply unit, ~~and whereby in~~ In the third shifting stage ~~[[,]]~~ the cooling circuit is supplied and at least two pump flows are conveyed to the ~~consumer~~ hydraulic-fluid-operated device from the hydraulic pressure supply unit. The shifting stages make it possible to use the valve apparatus, which can be actuated as needed, for also turning the cooling system on and off, for example of a clutch.

A further preferred embodiment of the hydraulic system is characterized in that the valve apparatus has a further shifting stage in which the cooling circuit is not supplied and a safety valve is ~~activated~~ operated. The safety valve can serve, for

example, to prevent overheating of the medium conveyed, or ~~represents~~ can represent a redundant opening mechanism for a clutch.

A further preferred embodiment of the hydraulic system is characterized in that the valve ~~apparatus~~, especially as a 2/2 way valve, is designed such that only one pump flow is conveyed from the hydraulic pressure supply unit to the ~~consumer~~ hydraulic-fluid-operated device as long as a first pressure, especially the adjustment pressure of an automatic transmission, is smaller than or equal to the sum of a second pressure, especially the contact pressure of an automatic transmission, and of a spring force, and in that at least two pump flows are conveyed from the hydraulic pressure supply unit to the ~~consumer~~ hydraulic-fluid-operated device if the first pressure, especially the adjustment pressure of an automatic transmission, is greater than the sum of the second pressure, especially the contact pressure of an automatic transmission, and of the spring force. In ~~this~~ that way, it is ~~guaranteed~~ assured that connection of at least one further pump flow will take place as a function of need.

A further preferred embodiment of the hydraulic system is characterized in that the valve ~~apparatus~~ includes a ~~tappet~~ spool whose one end face is acted upon ~~with~~ by the first pressure and whose other end face is acted upon by the second pressure and the spring force. The ~~tappet~~ spool moves as a function of the forces acting upon it and thus releases a flow path for one or more pump flows.

A further preferred embodiment of the hydraulic system is characterized in that the valve ~~apparatus~~ performs even additional valve functions in addition to turning on or shutting off the pump flow (~~[[=]]~~ the first valve function) in that the valve ~~tappet~~ spool releases or closes openings on other control units. ~~These~~ Those additional valve

functions can, for example, be an application of pressure on a hydraulic clutch, or the application of pressure on the ~~cone~~ conical pulleys of ~~an infinitely~~ a continuously variable transmission. The coupling of the first valve function with a further valve function represents a cost advantage because, instead of two slides and two boreholes, only one need be manufactured or machined.

The coupling of the first valve function with a further valve function likewise represents a functional advantage if a ~~volume~~ volumetric flow requirement is ~~actuated~~ controlled through the further valve function whose coverage takes place in the same valve by turning on a pump flow. The coupling of the first valve function with a further valve function likewise represents a functional advantage when ~~these~~ those valve functions can take place in connection with different motions of the ~~tappet~~ spool, that is, in part independently. For example, the additional valve function can represent the application of pressure on a clutch for the second gear, and the ~~tappet~~ spool now permits connecting an additional pump flow ~~when the clutch is not activated~~ by a further motion of the ~~tappet~~ spool when the clutch is not actuated. ~~This~~ That control and especially further displacement of the ~~tappet~~ spool takes place in a known manner, for example by applying a small electronically controlled pressure to a an end face of the ~~tappet~~ spool.

A further preferred embodiment of the hydraulic system is characterized in that the valve ~~apparatus~~ includes at least two valves whose switching respectively brings about the conveyance of at least one of the pump flows to the ~~consumer~~ hydraulic-fluid-operated device. Both valves in each case assume even further functions, as described above, for example respectively subjecting a clutch to pressure.

A further preferred embodiment of the hydraulic system is characterized in that the two valves are connected in series. Switching from one of the two valves leads to at least one of the pump flows being conveyed to the ~~consumer~~ hydraulic-fluid-operated device. The at least one pump flow is conveyed back to the input side of the hydraulic pressure supply unit.

A further preferred embodiment of the hydraulic system is characterized in that a ~~volume~~ volumetric flow regulating valve is arranged between the output side of the pressure supply unit and the ~~consumer~~ hydraulic-fluid-operated device. The ~~volume~~ volumetric flow regulating valve serves to regulate the ~~volume~~ volumetric flow to the ~~consumer~~ hydraulic-fluid-operated device. The excess ~~volume~~ volumetric flow is conveyed back to the input side of the hydraulic pressure supply unit.

A further preferred embodiment of the hydraulic system is characterized in that connection/disconnection of individual pump flows takes place as needed. For example, fixed rotational speed thresholds can be defined, at which the switching takes place. Here care should be taken that the requisite need is still being covered when disconnecting a pump flow.

A further preferred embodiment of the hydraulic system is characterized in that the ratio between the individual pump flows is asymmetrical. ~~Due to this,~~ Thereby it is possible to convey three different ~~volume~~ volumetric flows with one hydraulic pressure supply unit.

A further preferred embodiment of the hydraulic system is characterized in that a first pump flow covers approximately a third and a second pump flow approximately two thirds of the overall conveyed flow of the hydraulic pressure supply unit. With a

corresponding control logic, both pump flows together can supply 100% or only one pump flow can supply 33% or 66% of the entire conveyed flow of the hydraulic supply unit, as needed.

A further preferred embodiment of the hydraulic system is characterized in that the hydraulic pressure supply unit includes a vane cell pump or an internal gear wheel pump.

A further preferred embodiment of the hydraulic system is characterized in that a hydraulic ~~resistor~~ resistance is arranged between the valve ~~apparatus~~ and the input of the hydraulic pressure supply unit ~~that includes an~~ . An injector pump ~~into which the vane cell pump~~ is incorporated in the vane pump. The injector pump is used to assure proper filling of the preferably mechanically driven pump at higher rotational speeds.

The ~~indicated objective~~ above-identified object is ~~accomplished~~ achieved in connection with an automatic transmission for motor vehicles through a previously described hydraulic system. The hydraulic system ~~of~~ in accordance with the invention can also, however, be used in the steering system or the anti-roll system.

BRIEF DESCRIPTION OF THE DRAWINGS

Further advantages, features and details of the invention become apparent through the description below, in which various embodiments are described in detail with reference to the drawings. There is shown:

Figure 1 a hydraulic circuit diagram of a hydraulic system ~~of~~ in accordance with an embodiment of the invention for controlling an automatic transmission;

Figure 2 a block diagram of an arrangement for controlling a dual-flow pump with a valve ~~apparatus~~ and an additional flow regulating valve;

Figure 3 a block diagram of an arrangement for controlling a dual-flow pump with switching dependent upon two different pressures;

Figure 4 a block diagram of an arrangement for controlling a dual-flow pump with a ~~volume~~ volumetric flow ~~conductor~~; and

Figure 5 a block diagram of an arrangement for controlling a dual-flow pump with two valves connected in series.

DESCRIPTION OF THE PREFERRED EMBODIMENTS

Contemporary automatic transmissions for passenger cars control the starting process, the ~~gear~~ transmission ratio change, and the ~~activation for~~ operation of the reversing ~~assembly~~ clutch for forward/reverse travel, as well as for cooling and lubricating hydraulically. A hydraulic pressure supply unit and a hydraulic control unit are necessary for ~~this~~ those purposes.

Up until now, largely single-flow pumps have been used for supplying pressure to automatic transmissions. The pumps are distinguished in that their conveyed volumes are ~~purely~~ directly proportional to the their rotational speed. ~~This~~ That is disadvantageous insofar as, in designing pump sizes, often extreme situations such as, for example, a rapid adjustment at low rotational speeds, are ~~decisive for the~~ designs design specific. In many other driving situations, the ~~volume~~ volumetric flow then made available is not necessary. The efficiency of ~~these~~ those pumps is not optimal since, for example, unnecessarily ~~much~~ high hydraulic power is generated by

the mechanically driven pump at maximum speed. Second, the hydraulic components used, such as, for example, the pump, are ~~e.g.~~ subjected to unnecessarily high stress at maximum pump speed.

A pump concept is ~~created through~~ provided by the present invention, in which the required ~~volume~~ volumetric flow is generated as a function of ~~need~~ demand. In ~~this that~~ way, the dissipated hydraulic ~~dissipation~~ power, as well as the ~~pump~~ stress on the pump, can be reduced. With the hydraulic system ~~of~~ in accordance with the present invention, it proves to be advantageous that the system pressure can be reduced, since at a high system pressure leakages as a rule increase. The ~~actuation~~ control of the individual pump flows is preferably ~~conducted~~ carried out in such a way that the net ~~volume~~ volumetric flow remains ~~identical~~ unchanged. Generating less ~~volume~~ volumetric flow at the same rotational speed with cold oil than with warm oil is desirable, since at low oil temperatures less leakage is present and consequently the ~~volume~~ volumetric flow requirement drops.

It has proven to be advantageous if a portion of the ~~volume~~ volumetric flow made available by the pump is also used for cooling the transmission. With cold outside temperatures, it is advantageous to reduce the ~~volume~~ volumetric flow to the point that only the precisely required amount flows through the radiator, and consequently heat losses are reduced. With a very hot transmission, it is advantageous to generate more ~~volume~~ volumetric flow than needed in order to increase heat ~~output~~ transfer.

It has likewise proven to be advantageous if a portion of the ~~volume~~ volumetric flow furnished by the pump is also used for cooling individual components of the

transmission in danger of overheating, for example ~~[[,]]~~ friction clutches. At low friction output, load it is advantageous to shut the ~~volume~~ volumetric flow off, while at high friction output load (starting on a mountain), the required amount of cold oil is ~~peured~~ directed over the friction clutch to protect the latter from overheating.

A hydraulic control unit for an automatic transmission with ~~a driving~~ an input disk set 1 and an output disk set 2 is ~~represented~~ shown in Figure 1. The hydraulic system ~~represented~~ shown in Figure 1 ~~moreover also~~ serves to control a clutch 4 for reverse travel and a clutch ~~[[4]]~~ 5 for forward travel. ~~Activation~~ Actuation of the disk sets 1, 2 and the clutches 4, 5 takes place through a pump 8 in which a first pump flow 9 and a second pump flow 10 are generated parallel to each other. The two pump flows 9 and 10 are brought together through a ~~stop~~ check valve 12. An additional valve 14 serves to switch between the two pump flows 9, 10, such that either only pump flow 9 or both the pump flows 9, 10 are conveyed together in the direction of ~~consumer~~ hydraulic-fluid-operated devices 1, 2, 4, and 5.

A preferred embodiment for selective ~~actuation~~ control of two pump flows is ~~represented~~ shown in Figure 2. One pump 18, a vane ~~cell~~ pump, for example, is constructed and designed such that a first pump flow 19 is conveyed parallel to a second pump flow 20. The two pump flows 19 and 20 are connected with each other on the output side of the pump 18 through a conduit 22 in which a ~~stop~~ check valve 23 is arranged. The ~~stop~~ check valve 23 is arranged in the conduit 22 such that either only the first pump flow 19 or, ~~however,~~ the first pump flow 19 as well as the second pump flow 20 are conveyed through a conduit 24 to a ~~consumer~~ hydraulic-fluid-operated device 25. An orifice plate 26 is arranged in the conduit 24 between the

output side of the pump 18 and the ~~consumer~~ hydraulic-fluid-operated device 25. The orifice plate 26 ~~belongs~~, as is indicated through a dotted arrow 28, ~~to~~ is part of a flow regulating valve 29, which is arranged between the input side and the output side of the ~~consumer~~ hydraulic-fluid-operated device 25 in order to regulate the ~~volume~~ volumetric flow that is supplied to the ~~consumer~~ hydraulic-fluid-operated device 25. A 2/2 way proportional way valve is installed as the flow regulating valve 29. ~~It~~ If the ~~volume~~ volumetric flow conveyed to the ~~consumer~~ hydraulic-fluid-operated device 25 exceeds an adjustable maximum value, then the flow-regulating valve switches into its second position from the position ~~represented~~ shown in Figure 2. In its second position (not ~~represented~~ shown), the flow-regulating valve 29 releases a connection from the output side of the pump 18 through the conduit 24 past the ~~consumer~~ hydraulic-fluid-operated device to a ~~feedback~~ return conduit 30 that leads to a tank 31.

~~It is indicated by an~~ An arrow 32 indicates that the pressure in the ~~feedback~~ return conduit 30 serves to control an additional valve ~~apparatus~~ 34. The valve ~~apparatus~~ 34 is a 2/2 way proportional ~~way~~ valve that interrupts or releases (not ~~represented~~ shown) a connecting conduit 35 between the ~~feedback~~ return conduit 30 and the output side of the second pump flow 20 of the pump 18. The pressure in the ~~feedback~~ return conduit 30 is ~~dammed-up-in-front~~ restricted upstream of the valve ~~apparatus~~ 34 by a hydraulic ~~resistor~~ resistance 36.

With the hydraulic system ~~represented~~ shown in Figure 2, the valve ~~apparatus~~ 34 is used to convey selectively either only pump flow 19 or the two pump flows 19 and 20 together to the ~~consumer~~ hydraulic-fluid-operated device 25. The back pressure in the conduit 30 is used, as indicated by the arrow 32, to ~~switch~~ shift the

valve apparatus 34 against a spring. The hydraulic fluid ~~volume~~ volumetric flow flowing back over the ~~feedback~~ return conduit 30 into the tank 31, and/or toward the input side of the pump 18, meets the hydraulic ~~resistor~~ resistance 36, which generates the back pressure as a function of the ~~volume~~ volumetric flow. The injector pump used in vane cell pumps can, ~~for example,~~ be used as a hydraulic ~~resistor~~ resistance 36, ~~for example.~~ Such an injector pump is needed in vane cell pumps to assure proper filling of the pump at higher rotational speeds. The hydraulic ~~resistor~~ resistance 36 is ~~represented by a baffle~~ shown as an orifice plate in Figure 2.

The dual-flow pump 18 is ~~by way of example~~ driven by a crankshaft of an internal combustion engine, for example, and consequently conveys a hydraulic fluid ~~volume~~ volumetric stream as a function of pump rotational speed. The two pump flows 19 and 20 are brought together ~~through~~ by the ~~stop~~ check valve 23 and are supplied to the ~~consumer~~ hydraulic-fluid-operated device 25 through the conduit 24 ~~by~~ and the orifice plate 26 ~~in the position of~~ when the pilot control valve 34 ~~represented is in the position shown~~ in Figure 2. As for the ~~consumer~~ hydraulic-fluid-operated device, it can be, as ~~represented~~ shown in Figure 1, a disk set for adjusting the gear ~~ratios~~ transmission ratio condition or a clutch in an automatic transmission. Hydraulic fluid flowing back from the ~~consumer~~ hydraulic-fluid-operated device 25 through the ~~feedback~~ return conduit 30 is supplied to the pump 18 again through the hydraulic ~~resistor~~ resistance 36.

If the rotational speed of the pump 18 is increased, then more hydraulic fluid is ~~conveyed~~ needed and is fed to the ~~consumer~~ hydraulic-fluid-operated device 25. The return flow of hydraulic fluid ~~ever~~ through the ~~feedback~~ return conduit 30 to the suction

tract section of the pump 18 is correspondingly higher. The back pressure ~~on~~ increase at the hydraulic ~~resister~~ resistance 36 ~~increasing~~ owing to this that leads to the valve apparatus 34 shortcircuiting the second pump flow 20 of the pump 18 with the ~~back-flowing~~ return flow of hydraulic fluid in the ~~feedback~~ return conduit 30. In ~~this~~ that way, the pressure of the second pump flow 20 ~~declines~~ decreases to the back pressure in the ~~feedback~~ return conduit 30 and ~~closes~~ the ~~stop~~ check valve 23 closes. The second pump flow 20 of the pump 18 is then switched to recycling and needs only ~~still~~ to convey against the back pressure in the ~~feedback~~ return conduit 30, which would be present in any case. In ~~this~~ that way, the ~~output consumption of~~ power required by the pump 18 is reduced.

If the rotational speed of the pump 18 increases, and therewith of the ~~volume~~ volumetric flow fed to the ~~consumer~~ hydraulic-fluid-operated device 25 ~~increases~~, then the ~~volume~~ volumetric flow regulating valve 29 ~~in connection~~ connected with the orifice plate 26 restricts the ~~volume~~ volumetric flow to the ~~consumer~~ hydraulic-fluid-operated device 25. The hydraulic system ~~represented~~ shown in Figure 2 affords the advantage that the second pump flow 20 is only added when needed. If the ~~consumer~~ hydraulic-fluid-operated device 25 needs more hydraulic fluid, and consequently no or little hydraulic fluid is being passed back through the flow regulating valve 29 and the ~~feedback~~ return conduit 30 to the tank 31 or toward the input side of the pump 18, then the ~~volume~~ volumetric flow ~~declines~~ decreases due to the hydraulic ~~resister~~ resistance 36, and therewith the back pressure in the ~~feedback~~ return conduit 30. ~~This~~ That causes the valve apparatus 34 to close, and the hydraulic fluid ~~volume~~ volumetric flow of the second pump flow 20 to be conducted back to the ~~consumer~~ hydraulic-fluid-

operated device 25, as ~~represented~~ shown in Figure 2. ~~This~~ That ensures the increased need for hydraulic fluid. If the need of the ~~consumer~~ hydraulic-fluid-operated device 25 ~~declines~~ decreases again, then correspondingly more hydraulic oil is fed back, which leads to a renewed switching of the second pump flow 20 to pressureless recycling ~~without pressure~~.

A ~~cutaway~~ portion of a hydraulic block diagram is ~~represented~~ shown in Figure 3 in which a valve apparatus for need-dependent ~~volume~~ volumetric flow generation is designated ~~with~~ by 37. The valve apparatus 37 includes a valve housing 38 that is connected with the output side of a pump ~~with~~ having two pump flows 41 and 42, through a conduit 39. The two pump flows 41 and 42 are connected with each other through a conduit 45 in which a ~~stop~~ check valve 46 is arranged. The fact that the output side of the pump flows 41 and 42 is connected with a ~~consumer~~ hydraulic-fluid-operated device (not ~~represented~~ shown) is indicated by an arrow 47. The input side of the pump flows 41 and 42 is connected with a hydraulic tank 49. A conduit 50 leads from the hydraulic tank 49 to the valve housing 38. A ~~tappet~~ spool 52 is ~~pre-stressed~~ biased by a spring 53 in the valve housing 38. The end face of the ~~tappet~~ spool 52 facing away from the spring 53 is acted upon ~~with~~ by a pressure P_1 \underline{P}_1 . The end face of the ~~tappet~~ spool 52 facing away from the spring 53 is acted upon ~~with~~ by a pressure P_2 \underline{P}_2 .

In the ~~state~~ condition of the valve apparatus 37 ~~represented~~ shown in Figure 3, the pump flow 42 is ~~conveyed in a circle~~ recirculated through the conduit 50 and does not reach the ~~consumer~~ hydraulic-fluid-operated device. Only the pump flow 41 reaches the ~~consumer~~ hydraulic-fluid-operated device through conduit 47. It is

assured ~~through~~ by the ~~step~~ check valve 46 that pump flow 41 does not reach the valve housing 38 through conduit 39. When the ~~tappet~~ spool 52 moves such that the conduit 39 is closed, the pressure at the output of the pump flow 42 rises until the ~~step~~ check valve 46 in the conduit 45 ~~releases~~ opens the connection to the pump flow 41. Then both pump flows 41 and 42 are conveyed to the ~~consumer~~ hydraulic-fluid-operated device. Connection of the pump flow 42 to pump flow 41 takes place when the product of a first constant with and adjusting pressure P_2 $\underline{P_2}$ is greater than the ~~sum from the~~ product of a second constant with and the sum of the contact pressure P_4 $\underline{P_1}$ and the force of the spring. The pressures P_4 $\underline{P_1}$ and/or P_2 $\underline{P_2}$ can alternatively be a pressure that controls a function requiring ~~volume~~ volumetric flow already present in the hydraulic control unit, or a pressure ~~generated~~ produced by an electric control apparatus controlled by a pilot valve.

The circuit diagram of a hydraulic system is ~~represented~~ shown in Figure 4 in which a first pump flow 56 as well as a second pump flow 57 of a vane ~~cell~~ pump are fed from a tank 55. The two pump flows 56 and 57 are connected with each other through a ~~step~~ check valve 58. The output of the first pump flow 56 is connected with a valve housing 62 of a valve ~~apparatus~~ 63 through a conduit 60. The second pump flow 57 is connected with the valve housing 62 through a conduit 61. A ~~tappet~~ spool 64 is moveably ~~accommodated~~ received in the valve housing 62 of the valve ~~apparatus~~ 63. The ~~tappet~~ spool can ~~assume~~ carry out even further functions in the left region (~~not represented~~ shown), in that oil channels ~~are~~ can be closed or opened according to the position of the ~~tappet~~ spool. A connection to a ~~consumer~~ hydraulic-fluid-operated device (~~not represented~~ shown) is indicated by an arrow 65 that

proceeds from the valve housing 62. Moreover a ~~feedback~~ return conduit 66 proceeds from the valve housing 62 and opens into the tank 55.

In the position of the ~~tappet~~ spool 64 ~~represented as shown~~ in Figure 4, the first pump flow 56 is conveyed over the conduits 60 and 65 to the ~~consumer~~ hydraulic-fluid-operated device. The second pump flow 57 is conveyed back into the tank 55 through conduit 61 and the ~~feedback~~ return conduit 66. ~~when~~ When the ~~tappet~~ spool 64 is moved to the right, the connection between the conduit 61 and the ~~feedback~~ return conduit 66 is interrupted, which leads to the pressure rising on the output side of the second pump flow 57 until the ~~stop~~ check valve 58 opens and the two pump flows 56 and 57 are conveyed together over conduits 60 and 65 to the ~~consumer~~ hydraulic-fluid-operated device.

A circuit diagram of a hydraulic system is ~~represented~~ shown in Figure 5 in which a first pump flow 71 as well as a second pump flow 72 are fed with hydraulic fluid from a tank 68. The two pump flows 71 and 72 are connected with each other through a ~~stop~~ check valve 73. A conduit to ~~consumer~~ hydraulic-fluid-operated devices (not ~~represented~~ shown) is indicated by an arrow 74. A connection between the output side of the second pump flow 72 to a valve housing 76 of a first valve is indicated by an arrow 75. A ~~tappet~~ spool 77 is ~~accommodated~~ displaceably moveably received in the valve housing 76. The valve housing 76 of the first valve is connected with a valve housing 79 of the second valve through a conduit 78. A ~~tappet~~ spool 80 is moveably ~~accommodated~~ received in the valve housing 79 of the second valve. The valve housing 79 of the second valve ~~stands in connection~~ is connected with the tank 68 through a ~~feedback~~ return conduit 81.

Each ~~In the left region not shown, each~~ of the valves represented ~~shown~~ in the regions ~~not represented shown on the left~~ can ~~assume~~ carry out even further functions with the same ~~tappet~~ spool in that oil channels are closed or ~~released~~ opened. For example, clutches can be ~~acted upon by pressure~~ pressurized, or in particular ~~consumer~~ hydraulic-fluid-operated devices requiring ~~volume~~ volumetric flow, such as a cooling system, can be switched on.

The second pump flow 72 is conveyed ~~over~~ through the conduits 75, 78, and 81 back into the tank 68, thus ~~in the circuit~~ recirculated, in the ~~position~~ positions of the ~~tappet~~ spools 77 and 80 in the associated valve housings 76 and 79 ~~represented as shown~~ in Figure 5. The connection between the two pump flows 71 and 72 is interrupted by the ~~stop~~ check valve 73. If one of the ~~tappet~~ spools 77 and 80 is moved to the right against the associated stop, then the ~~feedback~~ return flow into the tank 68 is interrupted. ~~This~~ That leads to a rise in pressure on the output side of the second pump flow 72. The pressure on the output side of the second pump flow 72 rises until the ~~stop~~ check valve 73 opens, and both pump flows 71 and 72 are jointly conveyed ~~over~~ through the conduit 74 to the ~~consumer~~ hydraulic-fluid-operated device. The two valves with valve housings 76 and 79 are thus ~~connected~~ switched in a row or in series.

The ~~patent~~ claims submitted with the application are formulation proposals without prejudice to obtaining further-reaching patent protection. The applicant reserves the right to claim additional feature combinations previously disclosed only in the description or the drawings.

References in the dependent claims refer back to the further development of the object of the main claim by the features of the respective dependent claim. They are not to be understood as a waiver of attaining an independent, objective protection for the feature combinations of the referred back dependent claims.

Since the objects of the dependent claims can form their own and independent inventions with respect to the state of the art on the priority day, the applicant reserves the right to make them the object of independent claims or divisional applications ~~for division~~. They can furthermore also contain independent inventions that have an independent configuration from the objects of the preceding dependent claims.

The embodiments are not to be understood as a restriction of the invention. Rather, numerous changes and modifications are possible in the framework of the present invention, especially such variants, elements, and combinations and/or materials, that can be inferred, for example, by combination or modification of individual features in combination with the general description and embodiments, as well as features described in the claims and contained in the drawings, or elements and procedural steps which can be inferred by the specialist with respect to accomplishing the objective and that lead by combinable features to a new object, or to new procedural steps, or to new procedural step sequences, also to the extent that they concern manufacturing, testing, and operating procedures.

Patent Claims

What is claimed is: